

```

% segway_eom.m
%
% "Segway-Style" Inverted Pendulum: Equations of Motion.
% (See notes from Lecture 13 for a description of the system.)
% Katie Byl, 2012.

clear all; format compact % compact produces single-spaced output

% Define symbolic variables in matlab:
syms phiw thetab L mb Jb mw Jw Rw g b tau

% 1a. GC's (generalized coordinates), and their derivatives:
GC = [{phiw},{thetab}]; % Using ABSOLUTE angles here
dphiw = fulldiff(phiw,GC); % time derivative. GC are variables (over time)
dthetab = fulldiff(thetab,GC);

% 1b. Geometry of the masses/inertias, given GC's are freely changing...
xw = Rw*phiw;
xb = xw+L*sin(thetab);
yw = 0;
yb = L*cos(thetab);

% 1c. Define any required velocity terms (for masses):
dxw = fulldiff(xw,GC);
dxb = fulldiff(xb,GC);
dyb = fulldiff(yb,GC);

% 2. Kinetic Energy:
T = (1/2)*(mw*dxw^2 + Jw*dphiw^2 + mb*(dxb^2 + dyb^2) + Jb*dthetab^2)

% 3. Potential Energy:
V = mb*g*yb

% 4. Lagrangian:
L = T-V

% 5. EOMs:
eq1 = fulldiff(diff(L,dphiw),GC) - diff(L,phiw)
eq2 = fulldiff(diff(L,dthetab),GC) - diff(L,thetab);
eq2 = simplify(eq2)

% 6. Xi: non-conservative terms
Xi1 = tau - b*(dphiw-dthetab) % Motor torque tau, and back emf damping b
Xi2 = -tau + b*(dphiw-dthetab) % (equal and opposite to above)

% BELOW IS THE OUTPUT FROM THIS MATLAB SCRIPT:

```

Output from segway_eom.m :
(Katie Byl, 2012)

```
T =  
(Jb*dthetab^2)/2 + (Jw*dphiw^2)/2 + (mb*((Rw*dphiw + L*dthetab*cos(thetab))^2 + L^2*dthetab^2*sin(thetab)^2))/2 + L  
(Rw^2*dphiw^2*mw)/2  
V =  
L*g*mb*cos(thetab)  
L =  
(Jb*dthetab^2)/2 + (Jw*dphiw^2)/2 + (mb*((Rw*dphiw + L*dthetab*cos(thetab))^2 + L^2*dthetab^2*sin(thetab)^2))/2 + L  
(Rw^2*dphiw^2*mw)/2 - L*g*mb*cos(thetab)  
eq1 =  
- L*Rw*mb*sin(thetab)*dthetab^2 + d2phiw*(Jw + Rw^2*mb + Rw^2*mw) + L*Rw*d2thetab*mb*cos(thetab)  
eq2 =  
Jb*d2thetab + L^2*d2thetab*mb - L*g*mb*sin(thetab) + L*Rw*d2phiw*mb*cos(thetab)  
Xi1 =  
tau - b*(dphiw - dthetab)  
Xi2 =  
b*(dphiw - dthetab) - tau
```

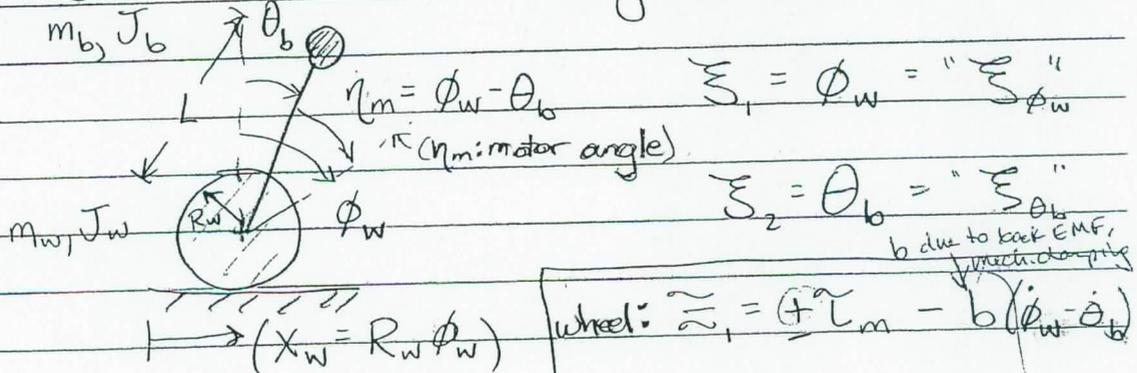
6.6 in Spong

Lecture 13

Controllability, Observability, State Space design

Segway example:

(neglects $n^2 J_{m, ...}$)



$\xi_1 = \phi_w = \xi_{\phi_w}$

$\xi_2 = \theta_b = \xi_{\theta_b}$

b due to back EMF, mechanical damping

wheel: $\ddot{\xi}_1 = +\tau_m - b(\dot{\phi}_w - \dot{\theta}_b)$

body: $\ddot{\xi}_2 = -\tau_m + b(\dot{\phi}_w - \dot{\theta}_b)$

$+\tau_m \rightarrow$ tends to $\uparrow \theta_b, \downarrow \phi_w$

sign depends on (arbitrary) leg wiring!

1) Geometry: Represent all masses

$y_w = 0$
 $x_w = R_w \phi_w$
 $x_b = x_w + L \sin \theta_b$
 $y_b = L \cos \theta_b$

for 2 wheels!

To include motor inertia
 $+\frac{1}{2} J_m (\dot{\phi}_w - \dot{\theta}_b)^2$
 (assume small)

2) Kinetic Energy:

$T^* = (\frac{1}{2} m_w \dot{x}_w^2 + \frac{1}{2} J_w \dot{\phi}_w^2) + \frac{1}{2} m_b (\dot{x}_b^2 + \dot{y}_b^2) + \frac{1}{2} J_b \dot{\theta}_b^2$

3) Potential Energy:

$V = m_b g y_b = m_b g L \cos \theta_b$

4) Solve for any required terms: $\dot{x}_w^2, \dot{x}_b^2, \dot{y}_b^2$

4) (cont'd)

$$\dot{X}_w = R_w \dot{\phi}_w$$

$$\dot{X}_b = R_w \dot{\phi}_w + L \cos(\theta_b) \dot{\theta}_b$$

$$\dot{Y}_b = -L \sin(\theta_b) \dot{\theta}_b$$

★!

$$\begin{aligned} \therefore \mathcal{L} &= \left(\frac{1}{2} m_w R_w^2 + \frac{1}{2} J_w + \frac{1}{2} m_b R_w^2 \right) \dot{\phi}_w^2 + \dots \\ &\quad \left(\frac{1}{2} m_b L^2 + \frac{1}{2} J_b \right) \dot{\theta}_b^2 + \dots \\ &\quad m_b R_w L \cos(\theta_b) \dot{\phi}_w \dot{\theta}_b + \dots \\ &\quad - m_b g L \cos(\theta_b) \end{aligned}$$

5) $\mathcal{L} = T^* - V$

$$\mathcal{L} = \frac{1}{2} m_w (R_w \dot{\phi}_w)^2 + \frac{1}{2} J_w \dot{\phi}_w^2 + \frac{1}{2} m_b \left((R_w \dot{\phi}_w + L \cos(\theta_b) \dot{\theta}_b)^2 + \dots \right) + \dots$$

$$+ \frac{1}{2} J_b \dot{\theta}_b^2 - m_b g L \cos(\theta_b) \quad \left(-L \sin(\theta_b) \dot{\theta}_b \right)^2$$

$$\mathcal{L} = \frac{1}{2} [m_w R_w^2 + J_w] \dot{\phi}_w^2 + \frac{1}{2} J_b \dot{\theta}_b^2 + \frac{1}{2} m_b \left(R_w^2 \dot{\phi}_w^2 + 2 R_w L \cos(\theta_b) \dot{\phi}_w \dot{\theta}_b + L^2 \dot{\theta}_b^2 \right) - m_b g L \cos \theta_b$$

6) EOM's

$$\#1 \quad \frac{d}{dt} \left(\frac{\partial \mathcal{L}}{\partial \dot{\phi}_w} \right) - \frac{\partial \mathcal{L}}{\partial \phi_w} = \tilde{\tau}_{\phi_w} = +T_m - b(\dot{\phi}_w - \dot{\theta}_b)$$

$(L^2 \cos^2(\theta_b) \dot{\theta}_b^2 + L^2 \sin^2(\theta_b) \dot{\theta}_b^2)$

$$\mathcal{L} = K_w \dot{\phi}_w^2 + K_b \dot{\theta}_b^2 + K_c \cos \theta_b \dot{\phi}_w \dot{\theta}_b - m_b g L \cos \theta_b$$

$$K_w \equiv \frac{1}{2} m_w R_w^2 + \frac{1}{2} J_w + \frac{1}{2} m_b R_w^2$$

$$K_b \equiv \frac{1}{2} m_b L^2 + \frac{1}{2} J_b$$

$$K_c \equiv m_b R_w L$$

$$\#1 \quad \frac{d}{dt} (2K_w \dot{\phi}_w + K_c \cos \theta_b \dot{\theta}_b) - 0 = \tilde{\tau}_{\phi_w}$$

$$\#2 \quad 2K_w \ddot{\phi}_w + K_c \cos \theta_b \ddot{\theta}_b - K_c \sin \theta_b \dot{\theta}_b^2 = \tilde{\tau}_{\phi_w} = +T_m - b\dot{\phi}_w + b\dot{\theta}_b$$

②

Eqn #2

$$\frac{d}{dt} \left(\frac{\partial \mathcal{L}}{\partial \dot{\theta}_b} \right) - \frac{\partial \mathcal{L}}{\partial \theta_b} = \tilde{\tau}_{\theta_b} = -T_m + b(\dot{\phi}_w - \dot{\theta}_b)$$

$$\tilde{\tau}_{\theta_b} \frac{d}{dt} \left(2K_b \dot{\theta}_b + K_c \cos \theta_b \dot{\phi}_w \right) = \left(-K_c \sin \theta_b \dot{\phi}_w \dot{\theta}_b + m_b g L \sin \theta_b \right)$$

$$2K_b \ddot{\theta}_b + K_c \cos \theta_b \ddot{\phi}_w - \cancel{K_c \sin \theta_b \dot{\theta}_b \dot{\phi}_w} + \cancel{K_c \sin \theta_b \dot{\phi}_w \dot{\theta}_b} - m_b g L \sin \theta_b = \tilde{\tau}_{\theta_b}$$

2

$$\therefore 2K_b \ddot{\theta}_b + K_c \cos \theta_b \ddot{\phi}_w - m_b g L \sin \theta_b = \tilde{\tau}_{\theta_b} = -T_m + b\dot{\phi}_w - b\dot{\theta}_b$$

Near equilibrium, $\left\{ \begin{array}{l} \sin \theta_b \approx \theta_b \\ \cos \theta_b \approx 1 \end{array} \right\} \left\{ \begin{array}{l} \dot{\theta}_b \approx 0 \\ \ddot{\theta}_b \approx 0 \end{array} \right\} \left\{ \begin{array}{l} \dot{\phi}_w \approx 0 \\ \ddot{\phi}_w \approx 0 \end{array} \right\}$
 (ignore H.O.T.)

#1 linearized

$$2K_w \ddot{\phi}_w + K_c \ddot{\theta}_b = +T_m - b\dot{\phi}_w + b\dot{\theta}_b$$

#2 linearized

$$2K_b \ddot{\theta}_b + K_c \ddot{\phi}_w - m_b g L \theta_b = -T_m + b\dot{\phi}_w - b\dot{\theta}_b$$

from 1: $\ddot{\theta}_b = \frac{1}{K_c} (-2K_w \ddot{\phi}_w + T_m - b\dot{\phi}_w + b\dot{\theta}_b)$

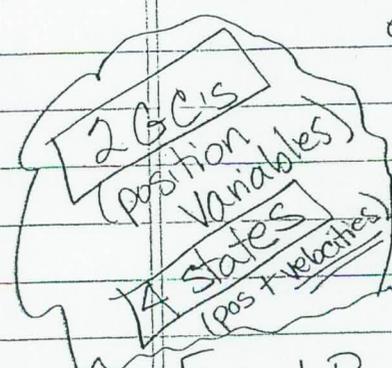
into 2: $2 \frac{K_b}{K_c} (-2K_w \ddot{\phi}_w + T_m - b\dot{\phi}_w + b\dot{\theta}_b) + K_c \ddot{\phi}_w - m_b g L \theta_b = -T_m + b\dot{\phi}_w - b\dot{\theta}_b$

UGH!! Is there a "clean" way to solve this??

Yes!
 • Can solve via MATRIX ALGEBRA:

Let us define all but highest order derivative of each Generalized Coordinate (GC) as a "state";

e.g. $m\ddot{y} + b\dot{y} + ky = F$
 \uparrow
 \dot{y} would be 2nd order deriv.



first state $x_1 = y$
 second state $x_2 = \dot{y}$

For 1P robot: $\begin{cases} x_1 = \phi_w \\ x_2 = \theta_b \\ x_3 = \dot{\phi}_w = \dot{x}_1 \\ x_4 = \dot{\theta}_b = \dot{x}_2 \end{cases}$ $X = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$

We wish to use a matrix formulation to describe the set of all possible states, the "state space", over time...
 i.e. to write the dynamic relationships (EOM).

$$M\dot{X} = A_m X + B_m u$$

More general goal is to write the (highest order) deriv's on LHS in terms of state variables!

matrix inverse

$$\dot{X} = A X + B u$$

M

$$(M^{-1}) M \dot{X} = (M^{-1} A_m) X + (M^{-1} B_m) u \quad (4)$$

From our equations:

$$\begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 2K_w & K_c \\ 0 & 0 & K_c & 2K_b \end{bmatrix} \begin{bmatrix} \dot{\phi}_w \\ \dot{\theta}_b \\ \ddot{\phi}_w \\ \ddot{\theta}_b \end{bmatrix} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 0 & -b & b \\ 0 & m_b g L & b & -b \end{bmatrix} \begin{bmatrix} \phi_w \\ \theta_b \\ \dot{\phi}_w \\ \dot{\theta}_b \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ K_m \\ -K_m \end{bmatrix} u$$

1) $\dot{\phi}_w \equiv \phi_w \rightarrow \frac{d}{dt}(\phi_w) = \dot{\phi}_w$, $\boxed{\dot{X}_1 = X_3}$

2) $\frac{d}{dt}(\theta_b) = \dot{\theta}_b$, $\boxed{\dot{X}_2 = X_4}$ ← no dots on RHS!
 ↑ all states have dots on LHS, only!

Eqm 1) 3) $2K_w \frac{d}{dt} \dot{\phi}_w + K_c \frac{d}{dt} \dot{\theta}_b = -b \phi_w + b \theta_b + K_m u$

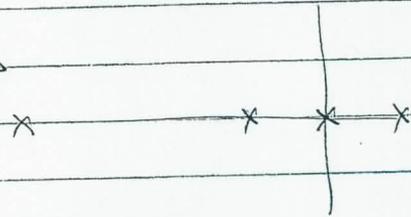
Eqm 2) 4) $K_c \frac{d}{dt} \dot{\phi}_w + 2K_b \frac{d}{dt} \dot{\theta}_b = m_b g L \theta_b + b \phi_w - b \theta_b - K_m u$

Form of $A = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & -a & -c & +c \\ 0 & +b & (\frac{g}{d}) & (\frac{g}{d}) \end{bmatrix}$, $B = \begin{bmatrix} 0 \\ 0 \\ f \\ L(-\frac{g}{d}) \end{bmatrix}$

$A \rightarrow \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & -650 & -270 & 270 \\ 0 & +230 & 70 & -70 \end{bmatrix}$ $B \rightarrow \begin{bmatrix} 0 \\ 0 \\ 280 \\ -74 \end{bmatrix}$

State Feedback Control

$\text{eig}(A) \rightarrow$ open-loop poles



Control Law is: $u = -Kx + r$

Then $\dot{x} = (A - BK)x$

Closed-loop Poles:

$\text{eig}(A - BK) \rightarrow$ need to be stable.

LQR - minimize a cost fn.

$$J = \int_{t=0}^{t=\infty} [x^T(t) Q x(t) + R u^2(t)] dt$$

$u^T R u$

Typically, Q is a diagonal matrix, giving penalty for each state error.

$$Q_{1,1} x_1^2 + Q_{2,2} x_2^2 + Q_{3,3} x_3^2 + Q_{4,4} x_4^2 + R u^2$$

in
Matlab

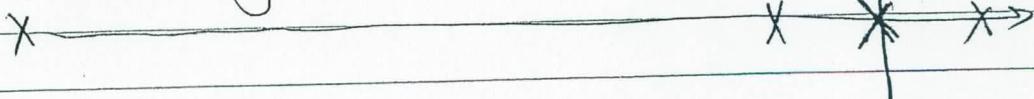
$$K_{\text{LQR}} = \text{lqr}(A, B, Q, R)$$

Form of A, B

$$A = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & -a_{32} & -a_{33} & +a_{33} \\ 0 & +a_{42} & \left(\frac{a_{33}}{d}\right) & \left(-\frac{a_{33}}{d}\right) \end{bmatrix}, B = \begin{bmatrix} 0 \\ 0 \\ b_{31} \\ \left(-\frac{b_{31}}{d}\right) \end{bmatrix}$$

Open Loop:

(MATLAB) $\text{eig}(A)$: open-loop poles



State feedback: Recall, $\dot{x} = Ax + Bu$. What is "u"?

- Linear control laws, where actuators get inputs that are a linear combination of the states:

$$u(t) = -Kx + r$$

\uparrow Gain matrix \leftarrow reference input

Example: Segway. One actuator. (2 wheels get same input...)

$$x \equiv \begin{bmatrix} \phi_w \\ \theta_b \\ \dot{\phi}_w \\ \dot{\theta}_b \end{bmatrix} \quad (4 \times 1), \quad K = [K_1 \quad K_2 \quad K_3 \quad K_4]$$

$r = 0$

$$\therefore u = -K_1 \phi_w - K_2 \theta_b - K_3 \dot{\phi}_w - K_4 \dot{\theta}_b = -Kx$$

$$\begin{aligned}\dot{x} &= Ax + Bu \rightarrow u = -Kx \\ &= Ax - BKx \\ &= (A - BK)x\end{aligned}$$

→ Now, instead of A , the dynamics are characterized by $(A - BK)$

MATLAB:

$\text{eig}(A - BK)$: closed-loop poles, 4th order, so 4 poles.
char. eqn can be written: $s^4 + \alpha_4 s^3 + \alpha_3 s^2 + \alpha_2 s + \alpha_1 = 0$

e.g., for 2nd order system: $(s - p_1)(s - p_2) = 0$

$$s^2 - (p_1 + p_2)s + p_1 p_2 = 0$$

$$s^2 + \alpha_2 s + \alpha_1 = 0$$

$$\alpha_2 = -p_1 - p_2 \quad \alpha_1 = p_1 p_2$$

Two (related) questions of interest:

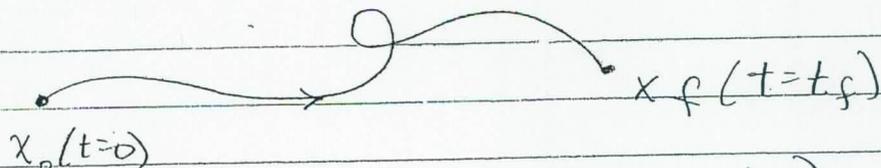
1) Given matrices A & B , can we find a control law, $u = -Kx$, to produce any arbitrary polynomial in s as a characteristic eqn?

i.e., can we set $\alpha_1, \alpha_2, \alpha_3$, etc.

{arbitrariness? i.e., can we place poles anywhere?}

$\text{eig}(A - BK) \leftrightarrow$ Pick value anywhere on s -plane?

(2) Can we take the system from any initial state, x_0 , to any final state, x_f , in a finite interval of time?



(- For non linear systems only Def'n #2 "makes sense" ...)
 → For a linear system, these are two ways of asking the SAME question!!!

"Is the system controllable?"

Answer: Yes, iff det $[B, AB, A^2B, \dots, A^{n-1}B] \neq 0$

Controllability matrix: $C \equiv [B, AB, A^2B, A^3B]$
sequency

if $\text{rank}(C) : \begin{cases} = n, & \text{then controllable} \\ < n, & \text{not controllable} \end{cases}$

$\text{rank}(C_{\text{sequency}}) = 4$

→ C , ^{more or less} shows how states can be affected at later times, due to an input now...

If system is controllable, to set ^(or "place") pole locations for the closed-loop (CL) system using MATLAB, can use either:

`place(A, B, pdes)`

(pdes cannot have "repeated" poles w/ multiplicity greater than # inputs in u)
(for place command)

or:

This is known as "pole placement"

`acker(A, B, pdes)`

not so accurate for higher-order systems ($n > 10$)
(for acker command)

Problems → Commanding arbitrary pole locations may require very large gains (∴ therefore saturate real-world actuator(s))!

Alternative "LQR" - linear quadratic regulator

Solves a linear quadratic optimal control problem.
[increasing R tends to reduce required u(t)]

$$J = \int_{t=0}^{t=\infty} [x^T(t) Q x(t) + u^T(t) R u(t)] dt$$

t=0 [for scalar $x \neq u$, $x^T Q x = Q x^2$, $u^T R u = R u^2$] "quadratic", as in SQUARED

Above, J is a cost function to minimize.

Optimal choice of gains, $K = K_{opt}$, minimizes this cost.

in MATLAB: `K = lqr(A, B, Q, R)` Q & R are typically DIAGONAL:
for Segway → $J = \int (Q_{11} x_1^2 + Q_{22} x_2^2 + Q_{33} x_3^2 + Q_{44} x_4^2 + R u^2) dt$