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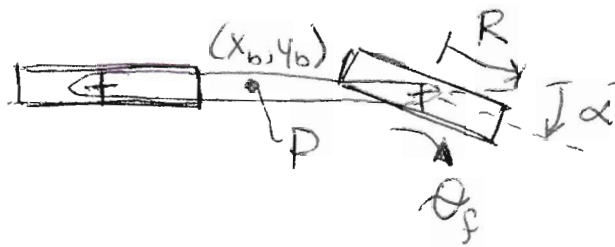


Fig. above shows phi (body angle) for a different config.

a) Sketch instantaneous center of rotation (ICR).

b) Write expressions for \dot{x}_b , \dot{y}_b and $\dot{\phi}_b$ for the configuration above, Either:

i. label needed dimensions

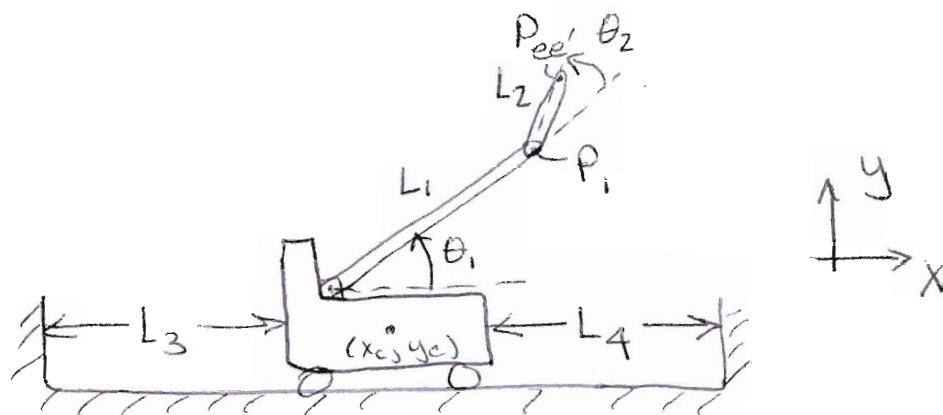
OR

ii. measure needed lengths by hand.

Expression should be a function of front wheel velocity, $d\theta_f/dt$, for geometry shown in the lefthand fig.

(HINT: Think of point P as it rotates about the ICR. Recall how Δy & Δx appear in the Jacobian...)

- ② The cart can roll such that $-L_3 \leq x_c \leq L_4$, $y_c = 0$, $0 \leq \theta_1 \leq \frac{\pi}{2}$ and $-\frac{\pi}{2} \leq \theta_2 \leq \frac{\pi}{2}$ for the mounted arm, with link lengths L_1 & L_2 , as shown.

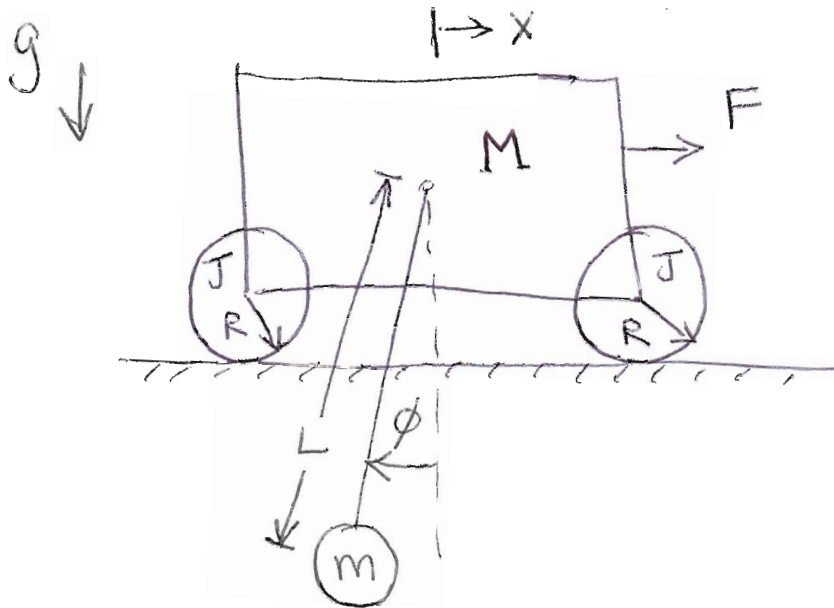


- Sketch the reachable workspace for end effector (at P_{ee}).
- Sketch the reachable workspace for P_1 (where L_1 & L_2 links meet).
- Sketch the dexterous workspace for P_{ee} , where $\theta_1 + \theta_2$ can be any angle.
- OK, that is a trick, since

$$-\frac{\pi}{2} \leq (\theta_1 + \theta_2) \leq \pi.$$

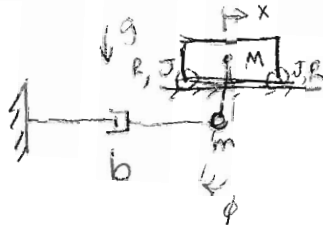
- Assume $0 \leq \theta_1 \leq \frac{\pi}{2}$ and θ_2 can be any angle. Now sketch the dexterous WS,

③ a) Derive the EOM for the system below, using the Lagrangian approach.



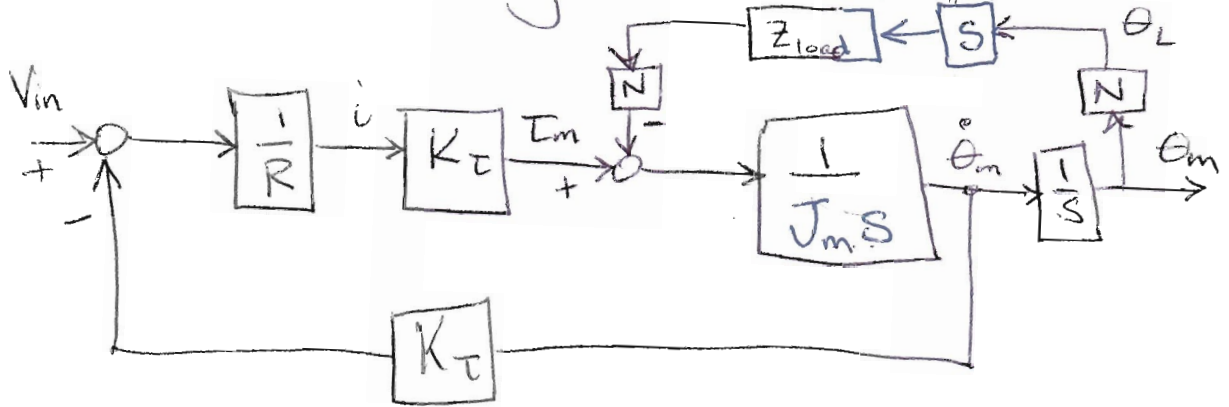
- Each wheel has radius R & inertia J . Assume no slip.
- Cart has mass M .
- Pendulum has length L & point mass m .
- There is gravity, as shown.
- Input F is applied to the cart.

b) Repeat when there is a damper between pendulum mass m and ground:



← For example, imagine air resistance is $-b\dot{x}_m$, where $x_m = x_M - L\sin\phi$

④ a) Reduce the following block diagram:



• Here Z_{load} is a mechanical impedance that is connected to a voltage-controlled motor with gear ratio.

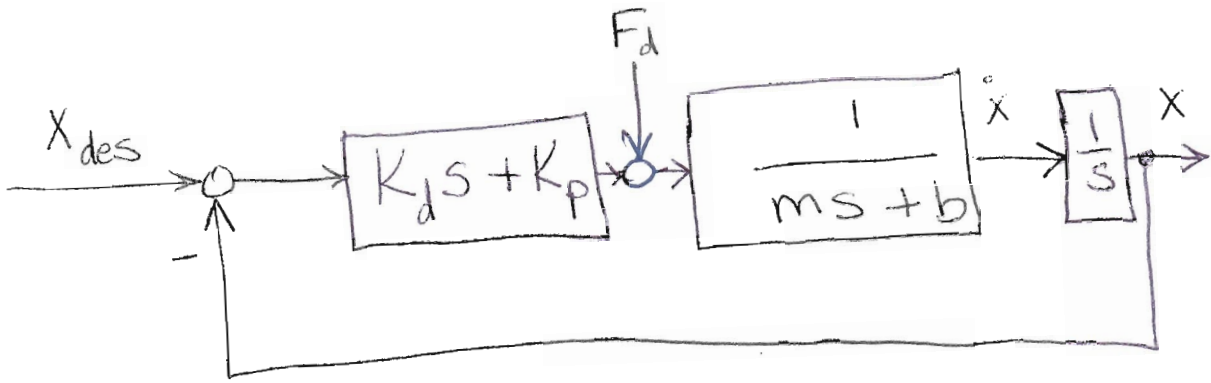
b) Solve for an impedance, Z_{load} , such that the entire system has second-order dynamics with $\omega_n = 10 \text{ rad/sec}$ and $\zeta = 0.2$

\uparrow (natural freq.) \uparrow (damping ratio)

* Use the following values, to make math easy:
 $K_T = 2 \left(\frac{\text{Nm}}{\text{Amp}} \right)$, $R = 1 \text{ (ohm)}$, $J_m = 1 \text{ (kg-m}^2\text{)}$, $N = 10$.

c) Is your solution for Z_{load} unique? Or are there multiple solutions?

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a) Solve for the closed-loop transfer function from X_{des} to X . (Hint, any other input, here F_d , is zero when calculating a "transfer function".)

b) What is the steady-state error for a unit step input in X_{des} ? ($F_d = 0$)

c) What is the transfer function from F_d to X ?

d) What is the steady-state value (error) for X given a unit step in F_d ? ($X_{des} = 0$)

e) What is required for the poles to be stable? (Assume $m > 0$, $b > 0$. What values can K_d & K_p have?)

f) What are the zeros of $X_{des}(s)$? (roots of the numerator)